The Finite Element Method Section 3

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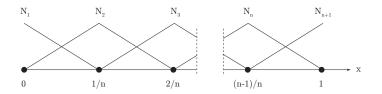
In the last section we showed how the solution to the 1-D Poisson Equation could be approximated using two degrees of freedom.

In this section we will:

- show how to move from a global to local description of the approximating functions
- introduce local descriptions of 'finite elements'
- introduce an implementation of a local-to-global assembly operator

Finite Elements: global description

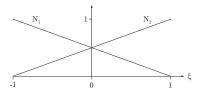
We have seen a 2-dof approximation of the 1-D domain. Accuracy of approximation may clearly be improved by increasing the number of degrees of freedom.



Finite Element: global description

Finite Elements: local description

We notice that the individual basis functions are highly localized in space and have the same repeated form. We can therefore standardize a single local element description.



Finite Element: local description

Local to Global Mapping

The local coordinate ξ is related to the global coordinate x via an affine transformation (i.e. a linear transformation followed by a translation)

$$\xi : [x_A, x_{A+1}] \to [\xi_1, \xi_2]$$

such that $\xi(x_A) = \xi_1$ and $\xi(x_{A+1}) = \xi_2$.

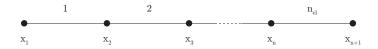
Visualizer

Determine $\xi(x)$ and $x(\xi)$ for the 2-dof finite element and consider the form of the shape functions

N.B. upper-case subscripts refer to the global element description, and lower-case subscripts refer to the local element description

Element 'Stiffness' Matrix and 'Force' Vector

Consider a finite element approximation to the 1D BVP with n_{el} elements and element numbers $e = 1, 2, \dots, n_{el}$.



Recall that we have previously defined the global 'stiffness' matrix $\mathbf{K} = [K_{AB}]$ and 'force' vector $\mathbf{F} = \{F_A\}$ with

$$\mathcal{K}_{AB} = a(\mathcal{N}_A, \mathcal{N}_B) = \int_0^1 \mathcal{N}_{A, \times} \mathcal{N}_{B, \times} dx$$

and

$$F_{A} = (N_{A}, I) + \delta_{A1}h - a(A_{A}, N_{n+1})g = \int_{0}^{1} N_{A}Idx + \delta_{A1}h - \int_{0}^{1} N_{A,x}N_{n+1,x}dxg$$

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Assembly: global sense

$$egin{aligned} \mathbf{K} &= \sum_{e=1}^{n_{el}} \mathbf{K}^e, \quad \mathbf{K}^e = [\mathcal{K}^e_{AB}] \ \mathbf{F} &= \sum_{e=1}^{n_{el}} \mathbf{F}^e, \quad \mathbf{F}^e = \{\mathcal{F}^e_A\} \end{aligned}$$

in which

$$\mathcal{K}^{e}_{AB} = a(\mathcal{N}_{A}, \mathcal{N}_{B})^{e} = \int_{\Omega^{e}} \mathcal{N}_{A,x} \mathcal{N}_{B,x} dx$$

and

$$F_A^e = (N_A, I)^e + \delta_{e1}\delta_{A1}h - a(A_A, N_{n+1})^e g$$

= $\int_{\Omega^e} N_A I dx + \delta_{e1}\delta_{A1}h - \int_{\Omega^e} N_{A,x}N_{n+1,x} dxg$

The elemental domain is defined as $\Omega^e = [x_1^e, x_2^e]$.

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Assembly: local sense

It is much more convenient to define an individual stiffness matrix and force vector for the eth element with respect to local node numbering:

$$\mathbf{k}^{e} = [k_{ab}] \qquad \mathbf{f}^{e} = [f_{a}]$$

$$k_{ab}^{e} = a(N_{a}, N_{b})^{e} = \int_{\Omega^{e}} N_{a,x} N_{b,x} dx$$

$$f_{a}^{e} = \int_{\Omega^{e}} N_{a} l dx + \begin{cases} \delta_{a1}h & e = 1\\ 0 & e = 2, 3, \cdots, n_{el} - 1\\ -k_{a2}^{e}g & e = n_{el} \end{cases}$$

These expressions may be determined from the global definitions by inspection.

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1-D Assembly Operator Implementation

We can obtain the global stiffness matrix and force vector from the locally-defined elemental stiffness matrix and force vector by defining an assembly operator $\mathbf{A}(\cdot)$ such that

$$\mathbf{K} = \mathop{\mathbf{A}}_{e=1}^{n_{el}} \left(\mathbf{k}^{e} \right) \qquad \mathbf{F} = \mathop{\mathbf{A}}_{e=1}^{n_{el}} \left(\mathbf{f}^{e} \right)$$

This operator may be implemented computationally for the 1-D case by means of a Location Matrix (LM) which has dimensions $n_{en} \times e$.

Global eqn. no.
$$A = LM(a, e) = \left\{egin{array}{cc} e & ext{if } a = 1 \\ e+1 & ext{if } a = 2 \end{array}
ight.$$

Visualizer

Example assembly operation for a 1-D problem approximated with 4 elements each having 2-dof

Summary

This section has introduced one of the most important aspects of the finite element method: the local element representation.

- Local element definitions are transformed to global coordinates via an affine mapping
- An assembly operator has been introduced which generates global ''stiffness' matrices and 'force' vectors from local element stiffnesses and forces
- A computational implementation of the assembly operator for a 1-D problem has been demonstrated

1-D problems are useful to consider for fundamental understanding, but it is now time to move to the use of the finite element for 2-D domains with applicability to 'real' engineering problems.

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